

Sistem Tiga-massa yang Tereksitasi Secara Parametrik

oleh

Dr. Siti Fatimah

sitifatimah@upi.edu

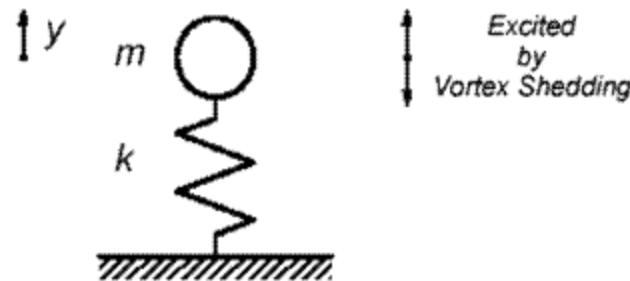
JURDIKMAT FPMIPA UPI BANDUNG

LATAR BELAKANG

Motivasi:

- Meredam atau mengurangi vibrasi yang sebabkan oleh arus induksi
- Studi terhadap sistem dua massa (two-degrees of problem system) yang self-excited dan tereksitasi secara parametrik menunjukkan dinamik yang kompleks dan menarik.

One-degree of freedom



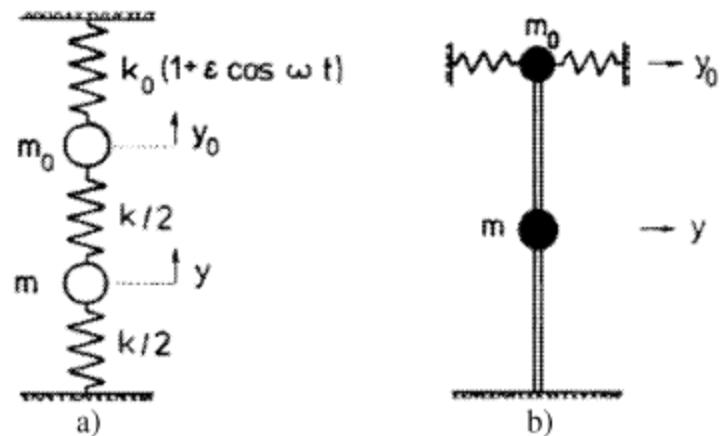
$$m \ddot{y} + b \dot{y} + k y = \gamma u ,$$

$$\ddot{u} + \alpha_0 \dot{u} + \omega_s^2 [u - \delta \operatorname{sgn}(\dot{y})] = 0 ,$$

Two-degrees of freedom

$$m \ddot{y} + [b - \beta_0 U^2 (1 + \gamma y^2)] \dot{y} + k y - \frac{1}{2} k y_0 = 0 ,$$

$$m_0 \ddot{y}_0 + b_0 \dot{y}_0 + k_0 (1 + \varepsilon \cos \omega t) y_0 - \frac{1}{2} k (y - y_0) = 0 .$$



Sumber: A. Tondl.. To The problem of Self-Excited Vibration Suppresion. *Engineering Mechanics*, Vol. 15, no. 4, p. 297-307, 2008.

Hasil Studi numerik dari *two-degrees of freedom system tereksitasi secara parametrik*

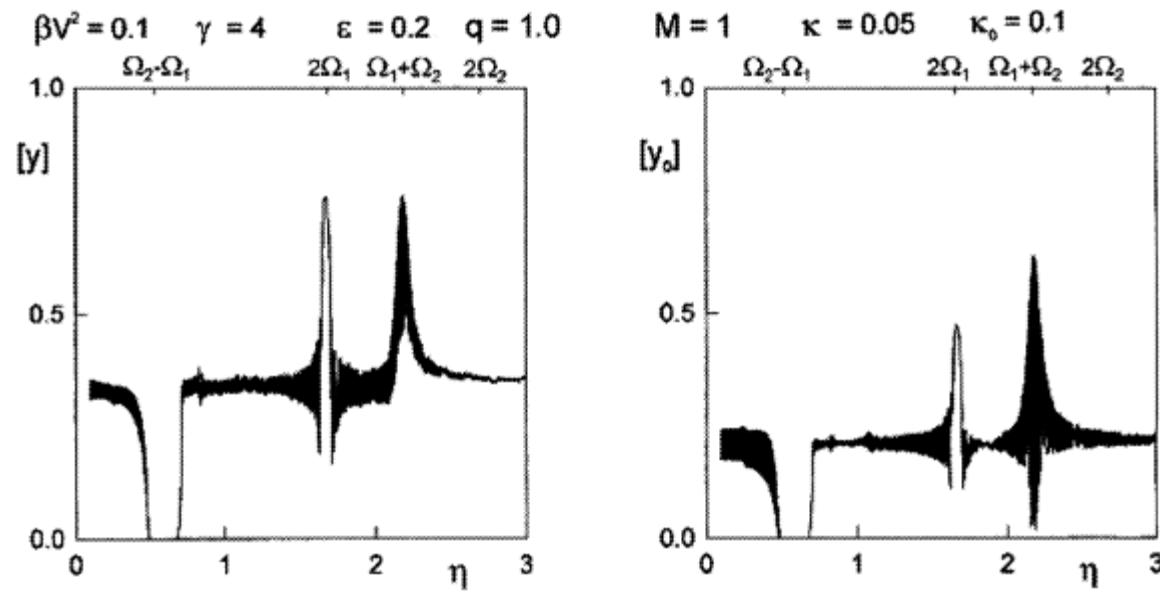


Fig.5: Extreme of y , y_0 in dependence on parametric excitation frequency ω

Hasil Studi Analitik: Dinamik pada *two-degrees of freedom* system tereksitasi secara parametrik (memuat persamaan Mathieu)

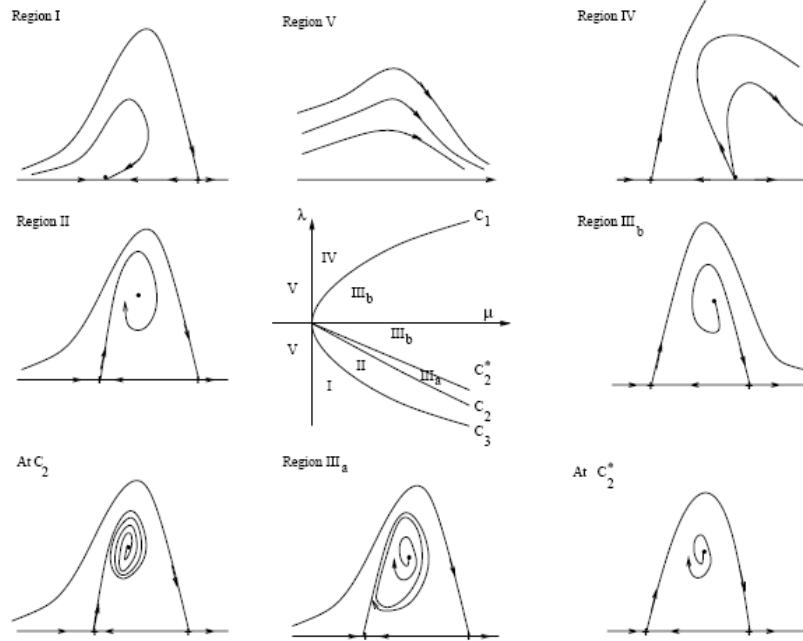


Figure 7: The phase-portraits of equation (71) in the (x, y) -plane for a specific value (μ, λ) in each region.

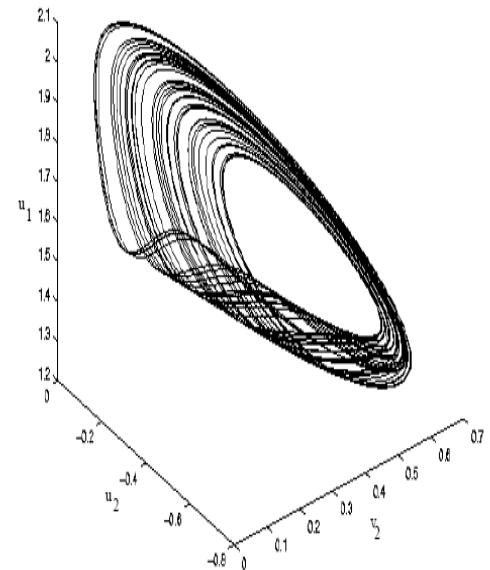


Figure 6: The strange attractor of the averaged system (68). The phase-portraits in the (u_2, v_2, u_1) -space for $c_2 < 0$ at the value $\sigma_2 = 5.3$. The Kaplan-Yorke dimension for $\sigma_2 = 5.3$ is 2.29.

Sumber: A. S.Fatimah and M. Ruijgrok. Bifurcations in an autoparametric system in 1:1 internal resonance with parametric excitation. *International J. Non-linear Mechanics*, 37(2), 297-308, 2002.

Hasil Studi Analitik: Dinamik pada two-degrees of freedom system (*memuat persamaan Rayleigh*)

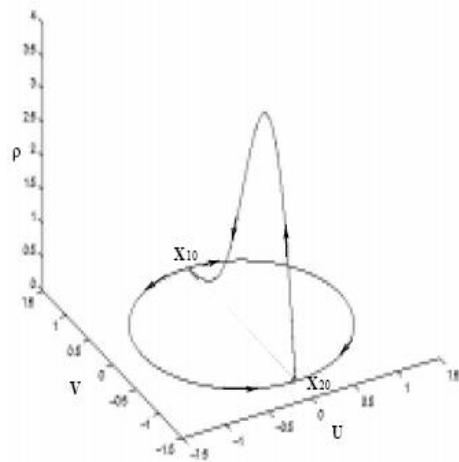


Figure 11: The robust heteroclinic cycle connecting the saddle points x_{10} and x_{20} .

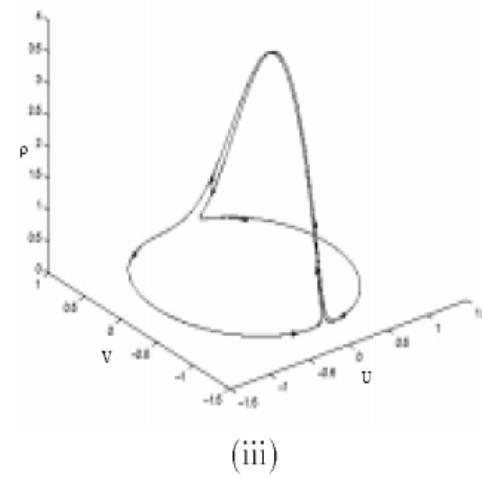
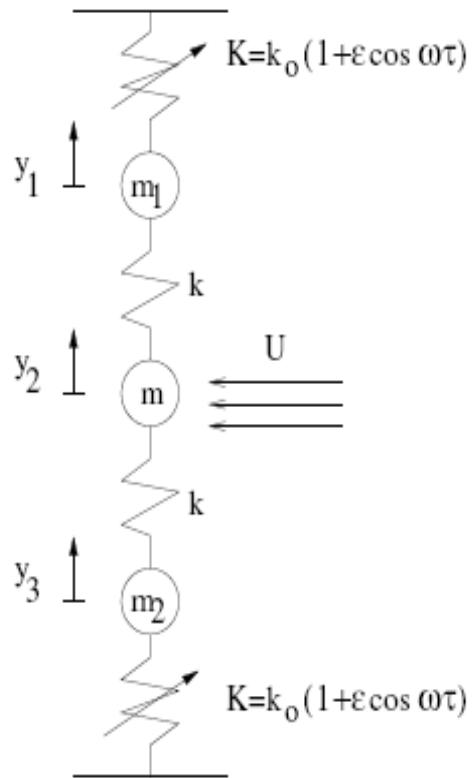


Figure 12: **Forced symmetry breaking.** (i) a long-periodic orbit for $\sigma = 0.01$, (ii) a long-periodic orbit for $\sigma = -0.01$, (iii) the combination of (i) and (ii).

Model tiga-massa



- Massa pusat m dan massa eksternal m₁ dan m₂.
- Konstanta pegas k
- Kecepatan gaya yg mengenai m =U, serupa persamaan Rayleigh
- Peredam berperiodik dg koefisisien K(t)
- K(t)=0 tidak ada pengaruh eksitasi parametrik

Persamaan Rayleigh:

$$\ddot{x} + x - \varepsilon(1 - \dot{x}^2)\dot{x} = 0$$

Persamaan Mathieu:

$$\ddot{x} + (1 + \varepsilon \cos \varpi t)x = 0$$

Sistem tiga-massa

$$\begin{aligned}m_1\ddot{y}_1 + b_0\dot{y}_1 + k_0(1 + \varepsilon \cos \omega t)y_1 + k(y_1 - y_2) &= 0 \\m\ddot{y}_2 - bU^2(1 - c\dot{y}_2)\dot{y}_2 + 2ky_2 - k(y_1 + y_3) &= 0 \\m_2\ddot{y}_3 + b_0\dot{y}_3 + k_0(1 + \varepsilon \cos \omega t)y_3 + k(y_3 - y_2) &= 0\end{aligned}\tag{1}$$

- Pada model di atas untuk kasus $m_1 = m_2 = m_0$ kemudian transformasikan dengan $\tau = \omega_0 t$ dimana $\omega_0 = \sqrt{2k/m}$ diperoleh

$$\begin{aligned}y_1'' + \varepsilon \hat{\kappa} y_1' + q^2(1 + \varepsilon \cos \eta \tau)y_1 + \frac{1}{2}M(y_1 - y_2) &= 0 \\y_2'' - \varepsilon \hat{\beta} V^2(1 - \gamma y_2'^2)y_2' + y_2 - \frac{1}{2}(y_1 + y_3) &= 0 \\y_3'' + \varepsilon \hat{\kappa} y_3' + q^2(1 + \varepsilon \cos \eta \tau)y_3 + \frac{1}{2}M(y_3 - y_2) &= 0\end{aligned}\tag{2}$$

Dengan $q^2 = k_0/m_0\omega_0^2$ $M = m/m_0$ $\eta = \omega/\omega_0$ $\kappa = \varepsilon \hat{\kappa} = b_0/m_0\omega_0$
 $\gamma = c\omega_0^2$, $V = U/U_0$

Analisis dari Sistem Tiga-massa yang Tereksitasi Secara Parametrik

- Transformasi sistem (2) dengan transformasi linear:

$$y_1 = x_1 + x_2 + x_3$$

$$y_2 = a_1 x_1 + a_2 x_2$$

$$y_3 = x_1 + x_2 - x_3$$

$$a_{1,2} = \frac{1}{M} \left(q^2 + \frac{1}{2}M - 1 \right) \pm \sqrt{\left(q^2 + \frac{1}{2}M - 1 \right)^2 + 2M}$$

- Diperoleh sistem yang memuat persamaan Mathieu berikut:

$$x_1'' + \Omega_1^2 x_1 + \varepsilon f_1(\mu_1, \cos \eta \tau, x_1, x'_1, x_2, x'_2, x_3, x'_3) = 0$$

$$x_2'' + \Omega_2^2 x_2 + \varepsilon f_2(\mu_2, \cos \eta \tau, x_1, x'_1, x_2, x'_2, x_3, x'_3) = 0$$

$$x_3'' + \Omega_3^2 x_3 + \varepsilon f_3(\cos \eta \tau, x_3, x'_3) = 0$$

$$f_1 = \theta_{11}x'_1 + \theta_{21}x'_2 + (Q_{11}x_1 + Q_{12}x_2)\cos\eta\tau + c_1x_1'^3 + c_2x_1'^2x_2' + c_3x_1'x_2'^2 + c_4x_2'^3$$

$$f_2 = \theta_{21}x'_1 + \theta_{22}x'_2 + (Q_{21}x_1 + Q_{22}x_2)\cos\eta\tau - c_1x_1'^3 - c_2x_1'^2x_2' - c_3x_1'x_2'^2 - c_4x_2'^3$$

$$f_3 = \hat{\kappa}x'_3 + q^2\cos\eta\tau x_3$$

- dengan

$$\Omega_{1,2}^2 = \frac{1}{2}\left(q^2 + \frac{1}{2}M + 1\right) \pm \sqrt{\frac{1}{4}\left(q^2 + \frac{1}{2}M + 1\right)^2 - q^2}$$

$$\Omega_3^2 = q^2 + \frac{1}{2}M$$

$$\theta_{11} = -\frac{1}{\alpha_1 - \alpha_2}(a_2\hat{\kappa} + a_1\hat{\beta}V^2) \quad \theta_{12} = -\frac{a_2}{\alpha_1 - \alpha_2}(\hat{\kappa} + \hat{\beta}V^2) \quad \theta_{21} = -\frac{a_1}{\alpha_1 - \alpha_2}(\hat{\kappa} + \hat{\beta}V^2)$$

$$\theta_{22} = \frac{1}{\alpha_1 - \alpha_2}(a_1\hat{\kappa} + a_2\hat{\beta}V^2) \quad Q_{11} = Q_{12} = -\frac{a_2}{\alpha_1 - \alpha_2}q^2 \quad Q_{21} = Q_{22} = \frac{a_2}{\alpha_1 - \alpha_2}q^2$$

$$c_1 = \frac{\alpha_1^3}{\alpha_1 - \alpha_2}\gamma\hat{\beta}V^2 \quad c_2 = \frac{3\alpha_1^2\alpha_2}{\alpha_1 - \alpha_2}\gamma\hat{\beta}V^2 \quad c_3 = \frac{3\alpha_1\alpha_2^2}{\alpha_1 - \alpha_2}\gamma\hat{\beta}V^2 \quad c_4 = \frac{\alpha_2^3}{\alpha_1 - \alpha_2}\gamma\hat{\beta}V^2$$

Kestabilan dari Solusi Trivial

- Keberadaan solusi trivial ditentukan oleh interval dengan kondisi berikut:

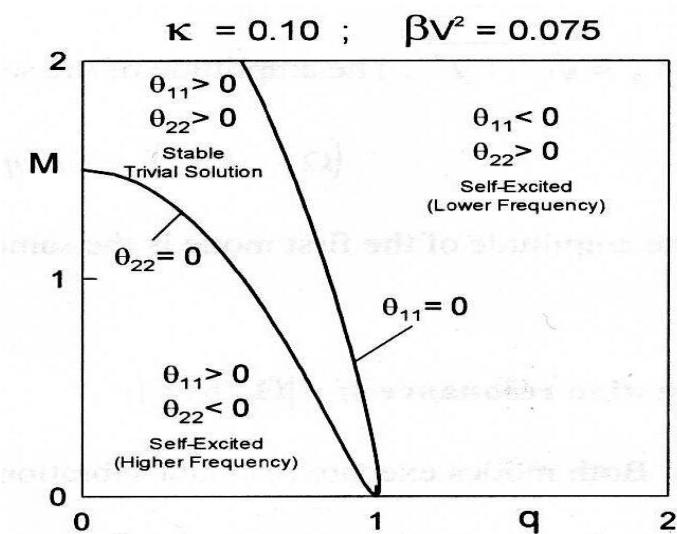
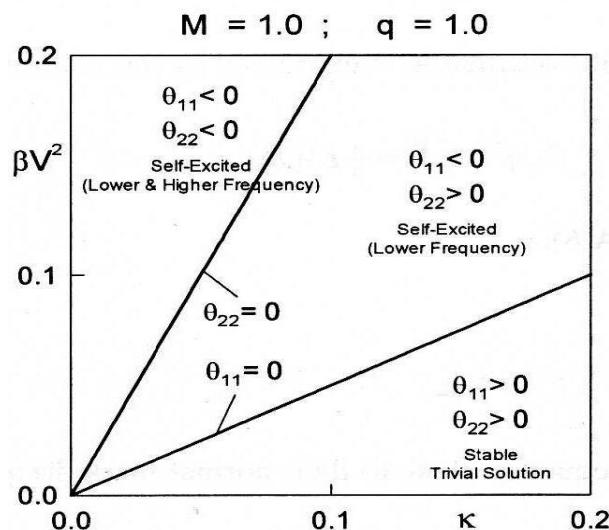
$$\theta_{11} + \theta_{22} > 0 \quad \text{atau} \quad k - \hat{\beta}V^2 > 0$$

- Batas kestabilan ditentukan oleh:

$$\eta_0 - \delta < \eta < \eta_0 + \delta$$

Dengan: $\eta_0 = \Omega_2 - \Omega_1$ dan $\delta = \varepsilon \frac{\theta_{11} + \theta_{22}}{\sqrt{|\theta_{11} \theta_{22}|}} \sqrt{-\frac{Q_{12} Q_{21}}{16 \Omega_1 \Omega_2} - \theta_{11} \theta_{22}}$

Grafik keberadaan solusi



$$M = 0.1 \quad q = 1$$

$$\kappa = 0 \quad \beta V^2 = 0.075$$

Solusi Non-trivial

- Pada sistem dengan parameter berperiodik, resonansi parametrik terjadi di sekitar nilai-nilai:
 -

$$\eta = \frac{2\Omega_j}{N} \quad j = 1, 2, \dots \quad N = 1, 2, 3, \dots$$

$$\eta = \frac{|\Omega_j + \Omega_k|}{N} \quad j, k = 1, 2, j \neq k$$

$$x_1(\tau) = R_1 \cos \Omega_1 \tau$$

$$x_2(\tau) = R_2 \cos \Omega_2 \tau$$

$$R_1 = \sqrt{-\frac{4\theta_{11}}{3c_1\Omega_1^2}} \quad R_2 = \sqrt{\frac{4\theta_{22}}{3c_4\Omega_1^2}}$$

- Untuk kasus $c_1 > 0$ dan $c_2 < 0$, solusi non trivial ada untuk
 - $\theta_{11} > 0$ dan $\theta_{22} < 0$

Fatimah, S and Verhulst, F (2003). *Suppressing flow-induced vibrations by parametric excitation*. Nonlinear Dynamics Journal, 31, 275-298.

Fatimah, S. (2004). *Redaman Vibrasi Arus Induksi dan Dinamik Solusi pada Persamaan van der Pol yang Terekstasi Secara Parametrik*. Laporan penelitian, Dana SP4.

Chow, S.N., and Hale, J.K. (1982). *Methods of Bifurcation Theory*. Appl. math.sciences, Springer-Verlag., New York.

Ecker, H. and Tondl, A. (2000). Suppression of flow-induced vibrations by a dynamic absorber with parametric excitation. *Proc. of 7th International Conference on Flow-induced Vibrations FIV2000*, Switzerland.

Fatimah, S. (2005). *Dinamika pada sistem Autoparametrik dengan Osilator*

memuat kombinasi dua buah Eksitasi Eksternal. Laporan penelitian, Dana SP4.

Guckenheimer, J.M and Holmes, P.J. (1990). Nonlinear Oscillations, dynamical systems and bifurcations of vector fields. *Appl.math.sciences*, Springer-Verlag, New York.

Kusumo, F. A. (2008). Analisis Sistem Konservatif yang Terpertubasi secara Singular. Disertasi, ITB.

Kuznetsov, Y.A. (1997). *Elements of Applied Bifurcation Theory*. Second edition, Springer, New York.

Sanders, J.A. and Verhulst, F. (1985). *Averaging Methods in Nonlinear Dynamical System*. Appl.math.sciences 59, Springer-Verlag, New York.

Tondl, A. (1991). *Quenching of self-excited Vibrations*, Elsevier, Prague.

Tondl, A. (1997). To the interaction of different types of excitation. In *Proc. of*

Sem. Interactions and Feedback 97, Prague, 25-26, 111-118, Prague.

Tondl, A., Kotek, V., and Kratochivil, C. (2001). *Vibration Quenching of Pendulum Type System by means of Absorbers*, CERM akademické, Chezh Republic.

Thompson, J.M.T. and Stewart, H.B. (2002). *Nonlinear Dynamics and Chaos*. sec. edition, John Wiley and Sons, Ltd, England.

Tuwankotta, J.M. (2006). Chaos in Coupled Oscillators with widely separated Frequencies and Energy-preserving nonlinearity. *Int. Journal on Nonlinear Mechanics*, 41, 180-191.

Wiggins, S. (1990). *Introduction to Applied Nonlinear Dynamical System and Chaos*. Springer-verlag, New York.